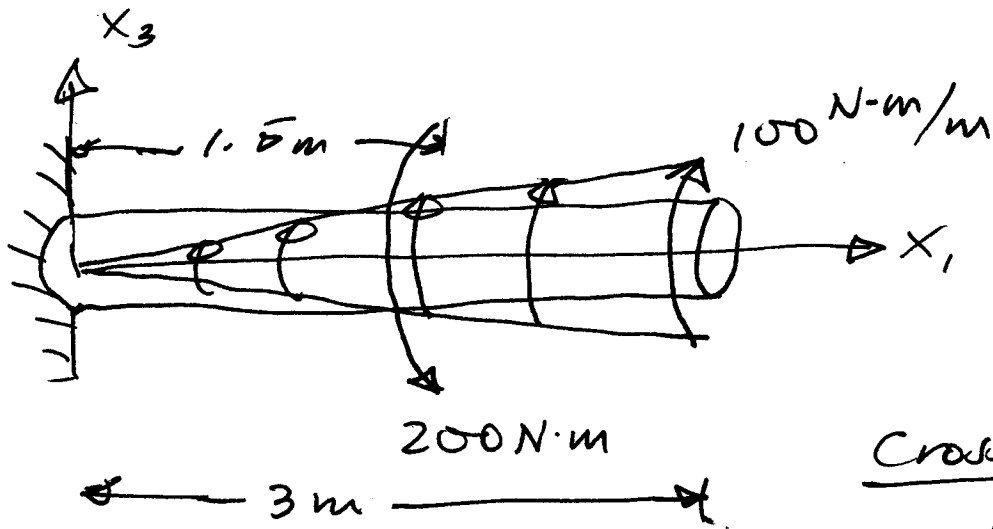


Unified Engineering Problem Set
Week 7 Spring, 2008

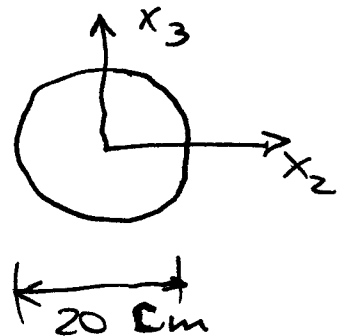
SOLUTIONS

M 7.1

Aluminum rod



Cross-Section



(a) Begin by determining equation for the distributed torque.

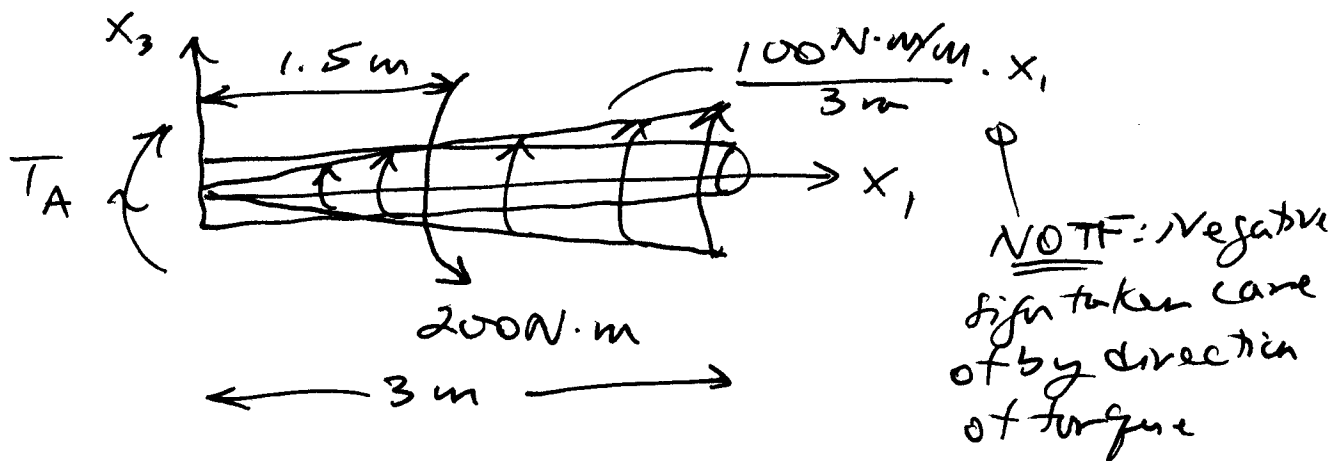
- it is negative
- it equals 0 at $x_1 = 0$
- it has intensity of $100 \text{ N}\cdot\text{m}/\text{m}$ at $x_1 = 3 \text{ m}$

This results in

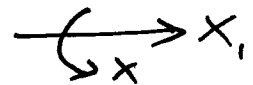
$$t(x_1) = - \frac{100 \text{ N}\cdot\text{m}/\text{m}}{3 \text{ m}} \cdot x_1 = -33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1$$

(NOTE: units are still torque intensity \Rightarrow correct)

\rightarrow Now draw Free Body Diagram



Take equilibrium of moments (torques) about x_1 . Use right hand rule (RHR) for +:



$$\sum T_{x_1} = 0 \quad \begin{matrix} \text{RHR} \\ \curvearrowright x_1 \\ + \end{matrix} \Rightarrow -T_A + 200 \text{ N}\cdot\text{m} + \int_0^{3\text{m}} 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1 \, dx_1 = 0$$

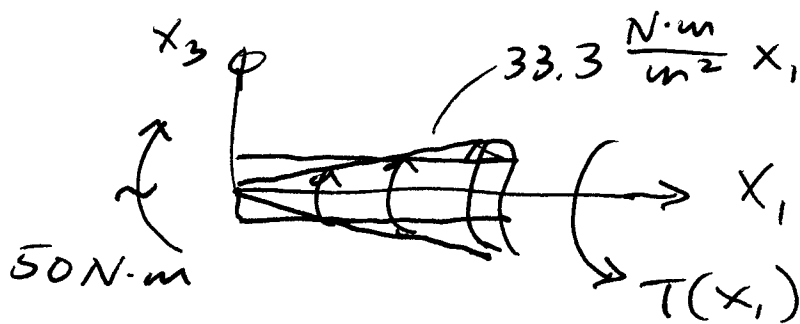
$$\Rightarrow T_A = 200 \text{ N}\cdot\text{m} - 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} \left(\frac{x_1^2}{2} \right) \Big|_0^{3\text{m}}$$

$$= 200 \text{ N}\cdot\text{m} - 150 \text{ N}\cdot\text{m}$$

Resulting in: $T_A = 50 \text{ N}\cdot\text{m}$

→ To determine the torque distribution, $T(x_1)$, make cuts in the shaft/rod in the regions of different loading.

First region $0 < x_1 < 1.5 \text{ m}$



Take equilibrium:

$$\sum T_{x_1} = 0 \quad \begin{array}{c} \text{RHR} \\ \leftarrow \\ \text{+} \end{array} \Rightarrow -50 \text{ N}\cdot\text{m} - \int_0^{x_1} 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1' dx_1' + T(x_1) = 0$$

$$\Rightarrow T(x_1) = 50 \text{ N}\cdot\text{m} + 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} \left(\frac{x_1'^2}{2} \right) \Big|_0^{x_1}$$

finally:

$$\boxed{T(x_1) = 50 \text{ N}\cdot\text{m} + 16.7 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^2}$$

$$0 < x_1 < 1.5 \text{ m}$$

OR

$$\text{use } \frac{dT}{dx_1} = -t(x_1)$$

$$= 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1$$

$$\begin{aligned}\Rightarrow T(x_1) &= \int 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1 dx_1 \\ &= 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} \left(\frac{x_1^2}{2} \right) + C,\end{aligned}$$

Use Boundary Condition that at $x_1 = 0$, T opposes the reaction torque $\Rightarrow T(0) = 50 \text{ N}\cdot\text{m}$

thus gives $C_1 = 50 \text{ N}\cdot\text{m}$

$$\Rightarrow T(x_1) = 16.7 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^2 + 50 \text{ N}\cdot\text{m}$$

$$0 < x_1 < 1.5 \text{ m}$$

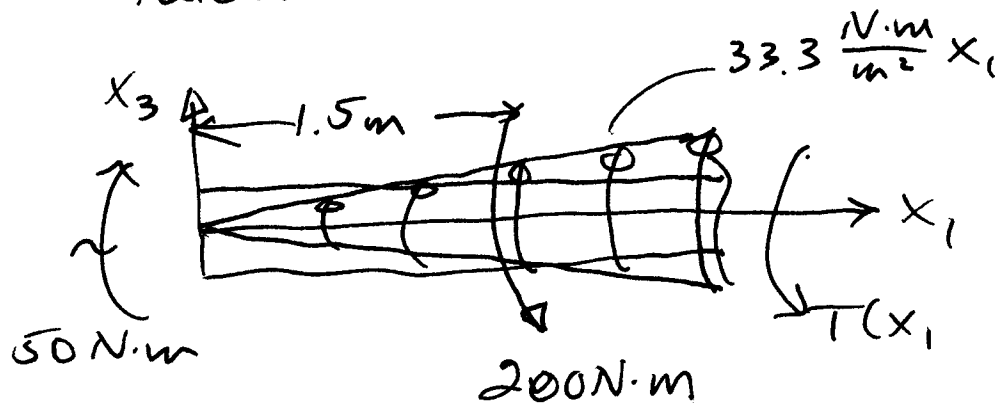
same result
(as it must be)

Proceed to

Second region

$$1.5 \text{ m} < x_1 < 3 \text{ m}$$

take a cut



Take equilibrium:

$$\sum T_{x_1} = 0 \quad \begin{matrix} \text{RAR} \\ \leftarrow \\ \rightarrow \\ + \end{matrix} \Rightarrow -50 \text{ N}\cdot\text{m} + 200 \text{ N}\cdot\text{m} - \int_0^{x_1} 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1' dx_1' + T(x_1) = 0$$

$$\Rightarrow T(x_1) = -150 \text{ N}\cdot\text{m} + 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} \left(\frac{x_1^2}{2} \right) \Big|_0^{x_1}$$

giving:

$$T(x_1) = -150 \text{ N}\cdot\text{m} + 16.7 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^2$$

$$1.5 \text{ m} < x_1 < 3 \text{ m}$$

OR use $\frac{dT}{dx_1} = -t(x_1)$

$$= 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1$$

again giving: $T(x_1) = 33.3 \frac{\text{N}\cdot\text{m}}{\text{m}^2} \left(\frac{x_1^2}{2} \right) + C_2$

Now need a boundary condition within this region. Could use the interface with region 1 or use the easier condition at the free end ($x_1 = 3 \text{ m}$) that there is no torque resultant: $T(3 \text{ m}) = 0$

$$\Rightarrow T(3 \text{ m}) = 0 = 150 \text{ N}\cdot\text{m} + C_2$$

$$\Rightarrow C_2 = -150 \text{ N}\cdot\text{m}$$

This results in:

$$T(x_1) = 16.7 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^2 - 150 \text{ N}\cdot\text{m}$$

$$1.5 \text{ m} < x_1 < 3 \text{ m}$$

again, same result

Now sketch the result...

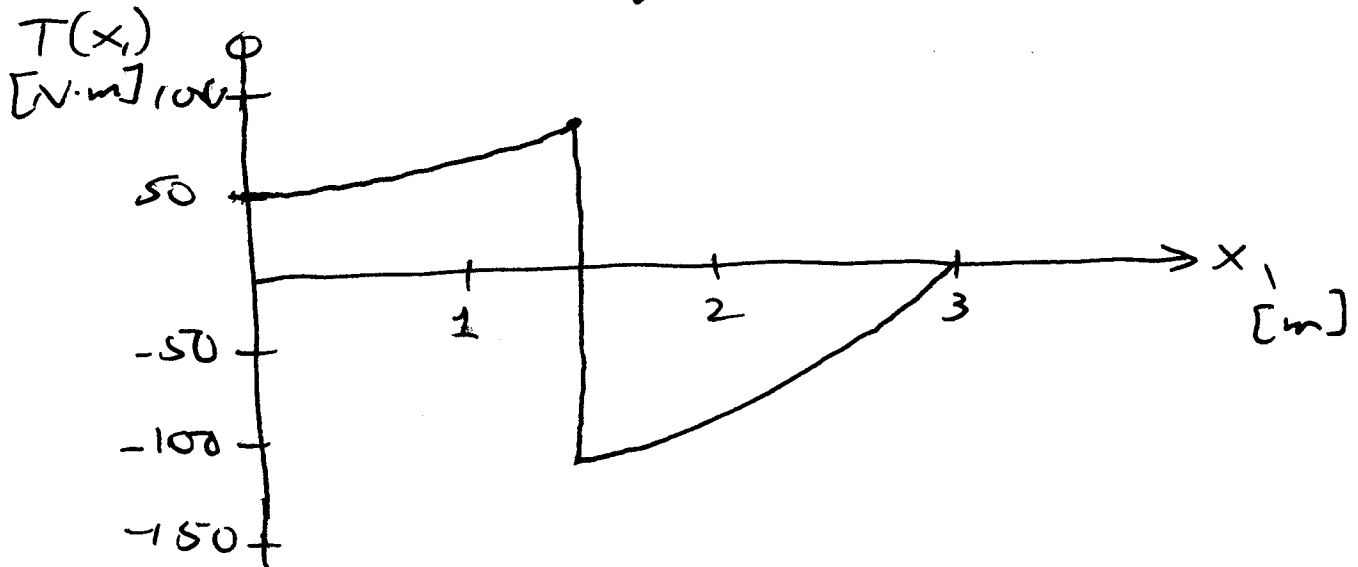
Note key points:

- (a) $x_1 = 0$, $T(x_1) = 50 \text{ N}\cdot\text{m}$
- (b) $x_1 = 1.5^- \text{ m}$, $T(x_1) = 87.5 \text{ N}\cdot\text{m}$
- (c) $x_1 = 1.5^+ \text{ m}$, $T(x_1) = -112.5 \text{ N}\cdot\text{m}$

(NOTE: Jump of $-200 \text{ N}\cdot\text{m}$... equal and opposite to applied point-torque)

- (d) $x_1 = 3 \text{ m}$, $T(x_1) = 0$

and $T(x_1)$ varies parabolic and positive in the two segments:



(b) To determine the twist of the rod, use the equation for the twist angle:

$$\frac{d\phi}{dx_1} = \frac{T}{GJ}$$

Consider each region starting at the point of the boundary condition on the twist angle:

$$\phi(x_1 = 0) = 0$$

So for Region 1:

$$\phi = \frac{1}{GJ} \int (50 \text{ N}\cdot\text{m} + 16.7 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^2) dx_1$$

$$\Rightarrow \phi(x_1) = \frac{1}{GJ} \left(50 \text{ N}\cdot\text{m} x_1 + \frac{16.7}{3} \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^3 \right) + C_3$$

$0 < x_1 < 1.5 \text{ m}$

using the Boundary condition gives:

$$C_3 = 0$$

So:

$$\phi(x_1) = \frac{1}{GJ} \left(50 \text{ N}\cdot\text{m} x_1 + 5.56 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^3 \right)$$

$0 < x_1 < 1.5 \text{ m}$

Proceed to Region 2 :

$$\text{Again using } = \frac{d\phi}{dx_1} = \frac{T}{GJ}$$

with $T(x_1)$ for this region:

$$\Rightarrow \frac{d\phi}{dx_1} = \frac{1}{GJ} \left(-150 \text{ N}\cdot\text{m} + 16.7 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^2 \right)$$

$$\text{So: } \phi = \frac{1}{GJ} \int \left(-150 \text{ N}\cdot\text{m} + 16.7 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^2 \right) dx_1$$

$$\text{giving: } \phi(x_1) = \frac{1}{GJ} \left(-150 \text{ N}\cdot\text{m} x_1 + \frac{16.7}{3} \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^3 \right) + C_4$$

Get a value to determine C_4 by matching the twist angle at the interface (boundary) with Region 1. From that result:

$$\phi(1.5 \text{ m}) = \frac{1}{GJ} (93.8 \text{ N}\cdot\text{m}^2)$$

matching gives:

$$\phi(1.5 \text{ m}) = \frac{1}{GJ} (93.8 \text{ N}\cdot\text{m}^2) = \frac{1}{GJ} (-206.2 \text{ N}\cdot\text{m}^2) + C_4$$

$$\Rightarrow C_4 = \frac{1}{GJ} (300 \text{ N}\cdot\text{m}^2)$$

So:

$$\phi(x_1) = \frac{1}{GJ} \left(-150 \text{ N}\cdot\text{m} x_1 + 5.56 \frac{\text{N}\cdot\text{m}}{\text{m}^2} x_1^3 + 300 \text{ N}\cdot\text{m}^2 \right)$$

$$1.5 \text{ m} < x_1 < 3 \text{ m}$$

So at the tip:

$$\phi(3\text{m}) = \frac{1}{GJ} (-450 \text{ N}\cdot\text{m}^2 + 150 \text{ N}\cdot\text{m}^2 + 300 \text{ N}\cdot\text{m}^2)$$

$$\Rightarrow \boxed{\phi(3\text{m}) = 0}$$

Could also determine GJ to get the torsional stiffness of the rod and values for the overall expressions of ϕ .

$$\text{In general: } J = \iint (x_2^2 + x_3^2) dA$$

We know that for a circular cross-section:

$$J = \frac{\pi R^4}{2}$$

$$\text{Here: } R = 10 \text{ cm} = 0.1 \text{ m}$$

$$\Rightarrow J = 1.57 \times 10^{-7} \text{ m}^4$$

Now find the shear modulus. We are given $E = 67 \text{ GPa}$, $\nu = 0.3$ for aluminum.

For isotropic materials:

$$G = \frac{E}{2(1+\nu)} \Rightarrow G = \frac{67 \text{ GPa}}{2(1+0.3)} = 25.8 \text{ GPa}$$

$$\oint GJ = (25.8 \times 10^9 \frac{\text{N}}{\text{m}^2}) (1.57 \times 10^{-7} \text{ m}^4) = 4.05 \times 10^6 \text{ N}\cdot\text{m}^2$$

(c) To find the shear stress, use:

$$\tau_{res} = \frac{T r}{J}$$

To determine the maximum magnitude, find the maximum magnitude of $T(x_1)$ and of r :

→ The value of r is maximized along the outer surface of the rod ($r = 10 \text{ cm}$)

→ $T(x_1)$ has a maximum magnitude of $-112.5 \text{ N}\cdot\text{m}$ at $x_1 = 1.5 \text{ m}$

Converting to consistent units:

$$\tau_{res} = \frac{(-112.5 \text{ N}\cdot\text{m})(10 \times 10^{-2} \text{ m})}{J}$$

from (b) calculated $J = 1.57 \times 10^{-4} \text{ m}^4$

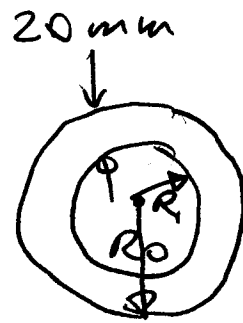
$$\text{So: } \tau_{res} = \frac{(-112.5 \text{ N}\cdot\text{m})(10 \times 10^{-2} \text{ m})}{1.57 \times 10^{-4} \text{ m}^4}$$

⇒ maximum magnitude is: (and is negative in direction)

$$\tau_{res} = 71.7 \times 10^3 \frac{\text{N}}{\text{m}^2} = 0.072 \text{ MPa}$$

at $r = 0.1 \text{ m}$
 $x_1 = 1.5 \text{ m}$

(d) Rod is now a hollow tube with the same outer radius and a wall thickness of 20 mm:



$$R_o = 10 \text{ cm}$$

$$R_i = 10 \text{ cm} - 20 \text{ mm}$$

$$= 8 \text{ cm}$$

The only thing that changes from the solid rod case is the cross-sectional polar moment of inertia (Shear Modulus stays the same - so

→ (a) Torque distribution Does not change

This configuration is statically determinate and does not depend upon the cross-sectional properties.

→ (b) All remains the same in considering the twist except J changes.

Can calculate the new J via
superposition (remove inner section):

$$J = \frac{\pi R_o^4}{2} - \frac{\pi R_i^4}{2} = \frac{\pi}{2} (R_o^4 - R_i^4)$$

$$= \frac{\pi}{2} (1^4 - 0.8^4) \text{ [m}^4\text{]} (\times 10^{-9})$$

$$\Rightarrow J = 0.927 \times 10^{-4} \text{ m}^4$$

This gives change for ϕ in general

$$\phi_{\text{new}} / \phi_{\text{old}} = J_{\text{old}} / J_{\text{new}} \quad \text{since } \phi \propto \frac{1}{J}$$

$$= \frac{1.57 \times 10^{-4} \text{ m}^4}{0.927 \times 10^{-4} \text{ m}^4}$$

$$\Rightarrow \frac{\phi_{\text{new}}}{\phi_{\text{old}}} = 1.69$$

$\Rightarrow \phi$ generally increases
by 69%

However $\phi_{\text{tip}} = 0$ so that value

does not
change

→ (c) The shear stress is $\tau_{res} = \frac{T r}{J}$

So only the polar moment of inertia, J , changes as T_{max} stays the same (via (a)) and the maximum value of r does not change.

So the change is via the inverse of J .

As shown in part (b)

maximum magnitude
 τ_{res} increases by 69%

location stays at

$$r = 0.1 \text{ m}$$

$$x_1 = 1.5 \text{ m}$$